EVALUATION REPORT OF KOPPERS COMPANY MAAG GEAR - WHEEL COMPANY LIMITED
30,000 HP AT 4000 RPM SELFSYNCHRONIZING OVERRUNNING CLUTCH

NAVSECPHILADIV PROJECT T-789 S-4622, TASK 14910

4 NOVE (BER, 1971

# Naval Ship Engineering Center, Philadelphia Division

PHILADELPHIA, PA. 19112

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# NAVSECPHILADIV PROJECT T-789

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#### ABSTRACT

A Koppers Company - Maag Gear Wheel Company Limited 30,000 HP at 4,000 RPM Self-Synchronizing Overrunning Clutch was tested at NAVSECPHILADIV. Due to gas turbine temperature limitations, maximum power transmission attained was 24,920 HP at 36,200 lb-ft of torque. Since this is essentially full torque, it is felt that the 30,000 HP transmission is well within the capabilities of this unit. Clutch synchronizing and engagement capability was confirmed on the steam turbine test facility by the accomplishment of seventy-seven smooth and trouble-free engagements at various input and output speeds and speed differentials.

The clutch is mechanically suitable for Naval use. Qualification for a particular application, however, would require that the operational profile for that application be specified and that additional testing be accomplished to determine reliability.

## SUMMARY PAGE

# PROBLEM

To test and evaluate the Koppers Company - Maag Gear Wheel Company Limited Self-Synchronizing Overrunning Clutch to determine its suitability for use in a combined Gas Turbine Power Plant.

# FINDINGS

No apparent pawl or ratchet wheel wear was evident after seventeen hours or ratchetting operation during which 1,000,000 ratchetting revolutions were accomplished. Due to ges turbine test facility limitations, the maximum power transmitted was 24,920 HP. This was successfully transmitted for fifty-five hours. Seventy-seven no load engagements were also successfully accomplished. This particular clutch design configuration considerably simplified clutch alignment, assembly, and disassembly.

# LIST OF ILLUSTRATIONS

FIGURE NO.	TITLE
1	Clutch Engaged and Disengaged Cross Sections
2	Clutch Hydraulic System
3	Steam Turbine Test Facility Machinery Arrangement and Rotational Inertias
4	Gas Turbine Test Facility Machinery Arrangement and Rotational Inertias
5	Steam Turbine Test Facility Vibration Transducer Locations
6	Clutch Speed Versus Time Curves
7	Pawl Tip Condition Before and After Engagement Testing
8	Pawl Condition Before and After Ratchetting Operation
9	Ratchet Wheel Condition Before and After Ratchetting Operation
10	Gas Turbine Test Facility Vibration Transducer Locations
11	Clutch Input Coupling Tooth Condition Prior to Full Power Engaged Operation
12	Clutch Input Coupling Tooth Condition Following



## DEPARTMENT OF THE NAVY

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NAVAL SHIP ENGINEERING CENTER, PHILADELPHIA DIVISION PHILADELPHIA, PA. 19112

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27 SEP 1974

From: Officer in Charge, Naval Ship Engineering Center

Philadelphia Division

To: Commander, Naval Ship Engineering Center (6145)

Subj: Evaluation of Koppers Company - Maag Gear - Wheel Company Limited 30,000 HP at 4000 RPM Self-Synchronizing Overrunning Clutch (S-4622 Task 14910) Final Report: Forwarding of

Ref: (a) Fonecon Mr. J. Cacciola, NAVSECPHILADIV 6730/ Mr. R. Peterson, NAVSEA 03413 of 13 Nov 1968

Encl: (1) Evaluation of Koppers Company - Maag Gear - Wheel Company Limited 30,000 HP, 4000 RPM Self-Synchronizing Overrunning Clutch

1. Enclosure (1), final report for NAVSECPHILADIV Project T-789, which was authorized by reference (a), is hereby forwarded.

E. L. LEWIS
By direction

Copy to:
NAVSEA 035(2),0932(4),03C
NAVSEC 6145D(2)
DDC (20)
Koppers Co., Baltimore, Md. (2)
NAVSECPHILADIV 6711F(2),6730,6733B(4)

#### REPORT OF INVESTIGATION

# Introduction

The increasingly widespread use of combined Gas Turbine Propulsion

Plants in Naval vessels has accentuated the need for a reliable clutching

system. A Koppers Company - Maag Gear Wheel Company Limited Self-Synchro
nizing Overrunning Clutch was tested to evaluate it for use in such systems.

It should be noted that this clutch was not tested for use in a particular vessel, nor were the test stands designed to simulate a particular power train. Both the steam turbine and gas turbine test stands had been previously constructed and were modified simply to accept the test clutch. On the steam turbine test stand the clutch was mounted between two steam turbines which allowed various input and output speeds. If this testing was successful, the clutch was installed on the gas turbine test stand and tested under load.

#### Object

The object of this test was to determine whether the capabilities of the subject clutch were in accordance with the following specifications:

a: Maximum Speed 4,400 rpm

Maximum Torque 39,400 lb. ft.

Full Power 30,000 HP

b. The clutch shall automatically engage when the input shaft speed accelerates through synchronism with the output shaft.

EVALUATION REPORT OF KOPPERS COMPANY MAAG GEAR - WHEEL COMPANY LIMITED 30,000 HP AT 4000 RPM SELF-SYNCHRONIZING OVERRUNNING CLUTCH

NAVSECPHILADIV PROJECT T-789 S-4622, TASK 14910

4 NOVEMBER, 1971

By

J. J. De Baecke

APPROVAL INFORMATION

Submitted By:

A. J. Brylen

Head, Machinery Systems Department

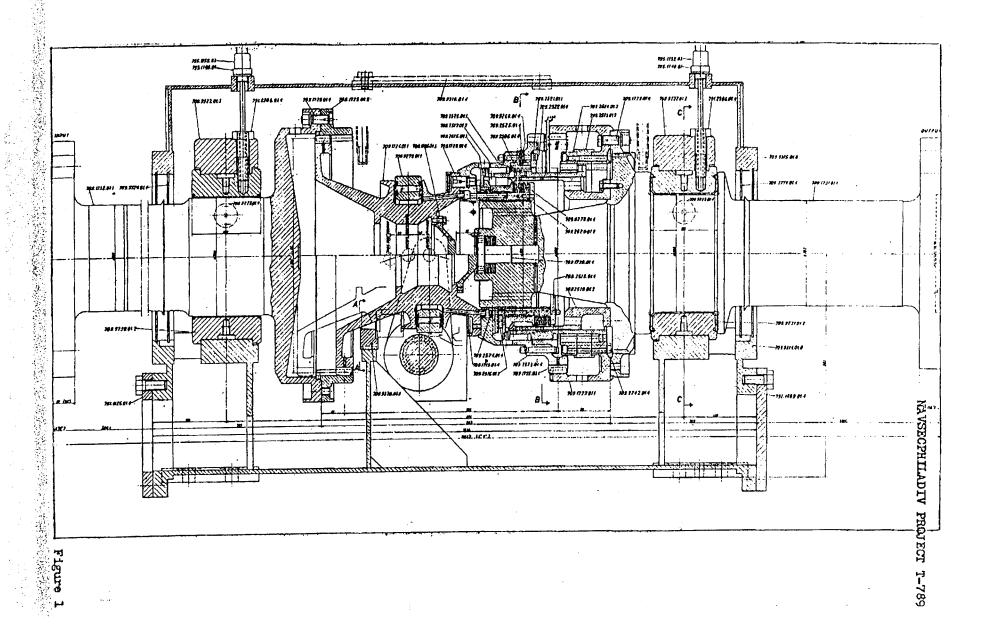
E. G. STORY
CAPTAIN, USN
Officer in Charge

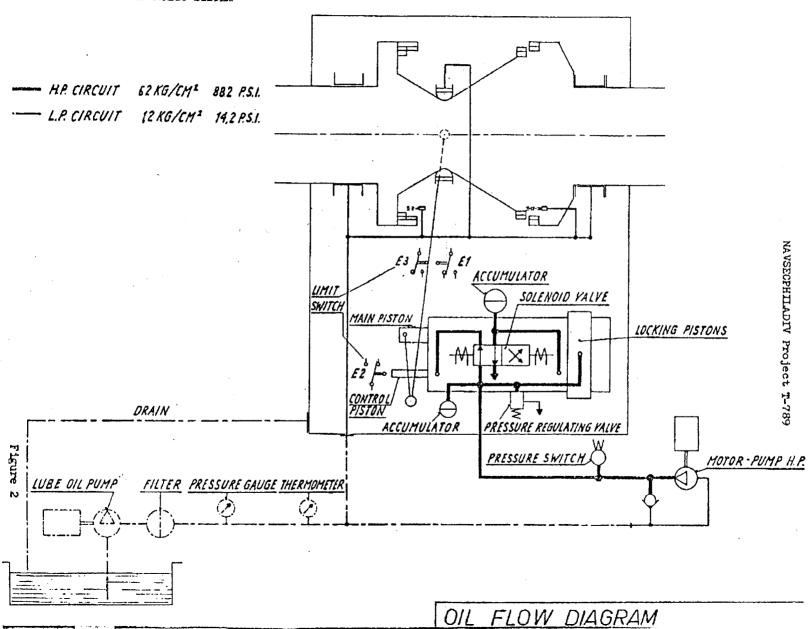
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# PROJECT T-789 TEST TIME FRAME

November 13, 1968	Telecon Mr. J. Cacciola, NAVSECPHILADIV Code 6730/ Mr. R. Peterson, NAVSHIPS 03413 Authorizing clutch Testing
August 1, 1969	Clutch received by NAVSECPHILADIV from Koppers Co.
November 10, 1969	Installation on Steam Turbine Test Facility commenced
January 21, 1970	Test operation on Steam Turbine Test Facility commenced.
February 10, 1970	Test operation on Steam Turbine Test Facility completed
April 17, 1970	Interim Test report submitted
October 19, 1970	Installation on Gas Turbine Test Facility commenced
December 12, 1970	Test operation on Gas Turbine Test Facility commenced
January 13, 1971	Test operation on Gas Turbine Test Facility completed
October 29, 1971	Final report submitted



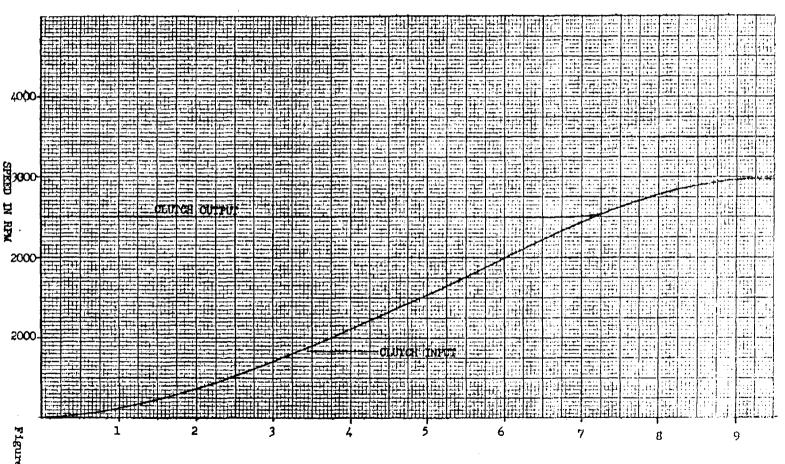


INPUT INERTIA 155.5 SLUG-FT<sup>2</sup>

OUTPUT INERTIA 95.0 SLUG-FT<sup>2</sup> (REFLECTED TO CLUTCH SHAFT)

MAYSECFHILADIV PROJECT 1-789

## CLUICH SPEED VERSUS TIME CURVE 2500 RPM DIFFERENTIAL ENGAGEMENT



TIME IN SECONDS

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